# Hydrodynamics of magnetic chains, and flexible fibres with a twist

Brennan Sprinkle April 30, 2020

# Outline

- 1. Problem set up
- 2. The model/method we used
- 3. Towards a new method

# **Colloidal Chains**

# Water









Experiments

# Simulation

Magnetic Model

Superparamagnetic particles (almost) instantaneously develop a magnetic moment  $\bar{m}$  aligned with the applied magnetic field B(t) so that

$$\bar{\boldsymbol{m}} \propto \boldsymbol{B}(t) = B_{\text{DC}} \hat{\boldsymbol{x}} + B_{\text{AC}} \sin(2\pi(50\text{Hz})t) \hat{\boldsymbol{y}} + B_{\text{AC}} \sin(2\pi(50\text{Hz})t) \hat{\boldsymbol{z}}.$$

In a chain of *N* beads, the magnetic force on bead *i* due to the interaction with bead *j* is

$$\boldsymbol{f}_{i,j}^{\text{mag}} = \frac{F_0}{(\boldsymbol{r}_{i,j}/a)^4} \left( 2\left(\boldsymbol{\bar{m}} \cdot \boldsymbol{\hat{r}}_{i,j}\right) \boldsymbol{\bar{m}} - \left( 5\left(\boldsymbol{\bar{m}} \cdot \boldsymbol{\hat{r}}_{i,j}\right)^2 - 1 \right) \boldsymbol{\hat{r}}_{i,j} \right), \quad (1)$$

where  $F_0 = 2.6 \times 10^{-12}$ N is the strength of the magnetic interactions.

We can define the 'angle'  $\beta$  of the magnetic field

 $\boldsymbol{B}(t) = B_{\text{DC}} \hat{\boldsymbol{x}} + B_{\text{AC}} \sin(2\pi(50\text{Hz})t) \hat{\boldsymbol{y}} + B_{\text{AC}} \sin(2\pi(50\text{Hz})t) \hat{\boldsymbol{z}}.$ 

as

$$\beta = \arctan\left(\frac{B_{\rm AC}}{B_{\rm DC}}\right).$$

And it was observed in experiments that the chains formed a helix shape for

$$\beta \approx 55^{\circ},$$

where stretching from  $B_{DC}$ , and twirling from  $B_{AC}$  are in balance.

Chain Model

- We maintain inextensibility between the centers of neighboring beads x<sub>i</sub> and x<sub>i+1</sub>
- The particles in the chain are bonded together so they shouldn't 'twist' much relative to each other.
- $\cdot$  The chain should be able to bend with modulus  $\kappa_b$

The chains in experiments clearly have a preferred chirality from a heterogeneity built into them during synthesis. We model this as a linearly decaying bending stiffness

$$\kappa_b(s) = \kappa_b^{\text{const.}} \frac{3+2(i/N)}{5}$$

**Kirchhoff Rods** 

# Particle orientation



## Kirchhoff Rods

Assume the frames at two neighboring beads only differ by a small rotation Let s be arclength along the chain.

 $\partial_{\mathsf{S}} \mathsf{Q} = \mathbf{\Omega} imes \mathsf{Q}$ 

Using

$$D^1 = D^2 \times D^3, \cdots$$

we get

$$\boldsymbol{\Omega} = \boldsymbol{\Omega}_1 \boldsymbol{D}^1 + \boldsymbol{\Omega}_2 \boldsymbol{D}^2 + \boldsymbol{\Omega}_3 \boldsymbol{D}^3 \tag{2}$$

$$= (D_{s}^{2} \cdot D^{3}) D^{1} + (D_{s}^{3} \cdot D^{2}) D^{2} + (D_{s}^{1} \cdot D^{2}) D^{3}$$
(3)

and discretize  $D_{s}^{k} \approx D_{i+1/2}^{k} - D_{i-1/2}^{k}$ 

#### Penalty Method

Lim and Peskin give forces and torques on the particles as

$$-f^{ch} = \overbrace{\partial_{s} \Lambda}^{(1)}$$

$$-\tau^{ch} = \partial_{s} \underbrace{\mathcal{M}(s)}_{(2)} + \underbrace{(\partial_{s} \mathbf{X}) \times \Lambda}_{(3)}$$

$$(4)$$

$$(5)$$

where  $\Lambda$  enforces that the triad be orthogonal and that  $D_i^3 = (\mathbf{x}_{i+1} - \mathbf{x}_i) \approx \partial_s \mathbf{x} = \mathbf{t}(s).$  $\Lambda = \Lambda_1 D^1 + \Lambda_2 D^2 + \Lambda_3 D^3$   $= \underbrace{\kappa_S (D^1 \cdot \mathbf{t}) D^1 + \kappa_S (D^2 \cdot \mathbf{t}) D^2}_{(a)} + \underbrace{\kappa_S (D^3 \cdot \mathbf{t} - 1) D^3}_{(b)}$ (6)
(7)

And

$$\mathbf{M}(s) = \left(\kappa_b \left[\Omega_1 \mathbf{D}^1 + \Omega_2 \mathbf{D}^2\right] + \kappa_T \Omega_3 \mathbf{D}^3\right)$$

#### The force and torque on bead *i* in the chain is

$$f_{i} = f_{i}^{ch} + \sum_{j>i} f_{i,j}^{mag}$$
(8)  
$$\tau_{i} = \tau_{i}^{ch} + \sum_{j>i} \tau_{i,j}^{mag}$$
(9)

hydrodynamics

# Equation of motion

Chains are small enough that steady Stokes equations govern hydrodynamics. Linearity of Stokes means that we can write

$$\begin{bmatrix} U_{1} \\ \vdots \\ U_{N} \end{bmatrix} = \mathcal{N}(\mathbf{x}_{1}, \cdots, \mathbf{x}_{n}) \cdot \begin{bmatrix} F_{1} \\ \vdots \\ F_{N} \end{bmatrix}, \quad U_{i} = \begin{bmatrix} u_{i} \\ \omega_{i} \end{bmatrix}, \quad F_{i} = \begin{bmatrix} f_{i} \\ \tau_{i} \end{bmatrix} \quad (10)$$
Mobility

•  $\mathcal{N}(x_1, \dots, x_n)$  is a 'Friction' or 'Mobility' operator that depends of the configuration of every particle in the chain and the geometry of the domain (e.g. bottom wall)

#### Update position we integrate $x_i, Q_i$ according to

$$\partial_t \mathbf{x}_i = \mathbf{u}_i,$$
 (11)

$$\partial_t \mathbf{Q}_i = \boldsymbol{\omega}_i \times \mathbf{Q}_i,$$
 (12) 14

# Details in Hydrodynamics

- Chains are small enough that hydrodynamics also must include terms which capture thermal fluctuations from the fluid. These are included but we won't really discuss them.
- Since the beads are so close together, we include lubrication corrections in the mobility

$$ar{\mathcal{N}} = \left( \mathcal{N}^{-1} - \mathcal{N}^{-1}_{\text{close particles}} + \left( \mathcal{N}^{\text{asymptotics}}_{\text{close particles}} \right)^{-1} 
ight)^{-1}$$

and steric repulsion to keep the beads from overlapping

# **Comparison With experiments**

Another Way

- 1. Unnecessary degrees of freedom (position and orientation of every particle in the chain)
- 2. Penalty formulation imposes a potentially large time step restriction

We should be able to model chains using only their unit tangent vectors  $\mathbf{t}_i$  and a scalar 'twist'  $\theta_i$  off of a reference axis (more on  $\theta_i$  in coming slides). Given a set of unit tangent vectors

$$\{\mathbf{t}_1, \mathbf{t}_2, \cdots, \mathbf{t}_N\}$$

we can find

$$x_i = x_0 + \sum_{j=1}^{i-1} t_i \approx x_0 + \int_0^s t(s') ds'$$

as a map from

$$t \in S^2 \rightarrow x \in \mathbb{R}^3$$

# **New Coordinates**



# **New Coordinates**

For a straight chain pointing *into* the page, we can define z as a reference axis and measure the local twist  $\theta_i$  off of this axis.



But what about a curved chain?

## Recall

$$\partial_{\mathsf{s}} t = \mathbf{\Omega} imes t$$

We can write

$$\Omega = m(s)t + t \times t_s$$
  
where  $m(s) = (D_s^1) \cdot D^2$  is the 'rate of twist'.

The material frame

$$Q(s) = [D^{1}(s), D^{2}(s), D^{3}(s)],$$

is to be determined by the physics of the problem.

But we can define a new 'intrinsic frame' (Bishop frame) so that the rate of twist vanishes.

$$B(s) = [t(s), u(s), v(s)]$$
such that (13)  
$$m_B(s) = u_s \cdot v = 0$$
(14)

and

$$\Omega_{B} = \underline{m}_{B}(s)t + t \times t_{s}$$

In the Bishop frame

$$\partial_{s} u = (t \times t_{s}) \times u$$
 (no twist)

which is integrable as

$$u(s) = \underbrace{\exp\left(s \ [t \times t_s]_{\times}\right)}_{p} u(0), \quad v(s) = t(s) \times u(s)$$

where **P** is a rotation matrix such that  $t(s + ds) = P \cdot t(s)$ , or a *parallel transport map*.

# **Bishop Frame**

Given t(s) and u(0), v(0) (which are arbitrary), the Bishop frame is completely determined. We use the bishop frame as a reference frame for measuring  $\theta$ .



# **New Coordinates**



## **Bishop Frame**

From the Bishop frame and  $\theta(s)$ , we compute the material frame as

$$D^{1} = \cos(\theta)u + \sin(\theta)v$$
 (15)

$$D^{2} = -\sin(\theta)u + \cos(\theta)v$$
 (16)

#### Hence

$$m(s) = (D_s^1) \cdot D^2 = \theta_s$$

and

$$\Omega = \underbrace{\theta_{\mathsf{S}}}_{\Omega_{\mathsf{S}}} t + \underbrace{t \times t_{\mathsf{S}}}_{\Omega_{\mathsf{1}} \mathsf{D}^{\mathsf{1}} + \Omega_{\mathsf{2}} \mathsf{D}^{\mathsf{2}}}$$
(17)



The energy functional for a constrained  $(x_{\rm s}=t)$  Kirchhoff rod is

$$E = \frac{1}{2} \int_{0}^{L} \kappa_{b} \left( \Omega_{1}^{2} + \Omega_{2}^{2} \right) + \kappa_{T} \Omega_{3}^{2} + \Lambda(s) \quad ds \quad (18)$$

$$= \frac{1}{2} \int_{0}^{L} \kappa_{b} \left( ||\mathbf{t} \times \mathbf{t}_{s}||^{2} \right) + \kappa_{T} \left( \theta_{s} \right)^{2} + \Lambda(s) \quad (19)$$

$$= \frac{1}{2} \int_{0}^{L} \kappa_{b} \underbrace{\left( ||\mathbf{t}_{s}||^{2} \right)}_{\text{Euler Beam}} + \kappa_{T} \left( \theta_{s} \right)^{2} + \Lambda(s) \quad (20)$$

We get the torque on out particles from *E* easily by varying theta

$$oldsymbol{ au} = -rac{\delta}{\delta heta} oldsymbol{ extsf{E}} = -\kappa_{ au} heta_{ extsf{ss}}$$

#### The force is a bit more nuanced. We vary the center-line $x(s) \leftarrow x(s) + \delta x(s)$ and compute

$$f = -\frac{\delta E}{\delta x} - \frac{\delta E}{\delta \theta_{\rm S}} \frac{\delta \theta_{\rm S}}{\delta x}$$

The quantity

# $\frac{\delta \theta_{\rm S}}{\delta {\bf X}}$

is tantamount to the variation of the Bishop frame along centerline

$$\frac{\delta\theta_{\rm s}}{\delta {\bf x}} = {\bf u}_{\rm s} \triangleleft \frac{\delta {\bf u}_{\rm s}}{\delta {\bf x}}$$

## Holonomy



 $\frac{\delta\theta_{s}}{\delta \mathbf{x}} = \mathbf{u} \angle \delta \mathbf{u} \approx ||\mathbf{t}_{s} \times \delta \mathbf{x}_{s}|| = (\mathbf{t} \times \mathbf{t}_{s}) \cdot \delta \mathbf{x}_{s} ds$ 

# Variation of the energy

$$\mathbf{r} = -\frac{\delta}{\delta\theta} \mathbf{E} = -\kappa_T \theta_{SS}$$
(21)  
$$\mathbf{f} = -\frac{\delta \mathbf{E}}{\delta \mathbf{x}} - \frac{\delta \mathbf{E}}{\delta\theta_S} \frac{\delta\theta_S}{\delta \mathbf{x}}$$
(22)  
$$= \kappa_b \mathbf{x}_{SSSS} - \kappa_T \left(\underbrace{\theta_S}_{\text{from } \frac{\delta \mathbf{E}}{\delta\theta_S}} \underbrace{(\mathbf{t} \times \mathbf{t}_S)}_{S}\right)_S + (\Lambda \mathbf{t})_S$$
(23)

#### Open question: How to do hydrodynamics with constrained chains and twist.